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# A novel slip-friction connector for seismic applications in timber structures: analytical and experimental investigation

Matteo Pellicciari<sup>a,\*</sup>, Angelo Aloisio<sup>b</sup>, Roberto Tomasi<sup>c</sup>, Lars Vidar Jakobsen Næsse<sup>c</sup>,  
Francesco Boggian<sup>c</sup>, Pasqualino Gualtieri<sup>b</sup>

<sup>a</sup>*DIEF, Department of Engineering “Enzo Ferrari”, via P. Vivarelli 10, 41125 Modena, Italy*

<sup>b</sup>*Department of Civil, Construction-Architectural and Environmental Engineering, Università degli Studi dell’Aquila, L’Aquila, 67100, Italy*

<sup>c</sup>*Faculty of Science and Technology, Norwegian University of Life Sciences, As, Norway*

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## Abstract

Slip-friction connectors are increasingly recognized as an effective strategy for enhancing the seismic performance of structures. Conventional solutions, however, commonly rely on bolt pretension to activate frictional resistance, which may lead to drawbacks such as preload losses over time, installation challenges, and concerns regarding long-term reliability. To overcome these limitations, this study proposes an innovative slip-friction connector that replaces bolt pretension with an elastic restoring force provided by a precompressed spring. The mechanical behavior of the proposed device is investigated through analytical modeling and experimental testing. An analytical solution is developed based on equilibrium equations, capturing the interaction between frictional sliding and elastic restoring effects in the force-displacement response. Cyclic tests are performed to validate the model and assess the energy dissipation capacity of the connector. Experimental results show excellent agreement with the analytical predictions, confirming the robustness of the proposed formulation. The findings of this study suggest that the proposed connector offers a promising solution for seismic applications in timber structures, with particular reference to cross-laminated timber (CLT) panels. In this context, the device may serve as an alternative to conventional connections, such as hold-downs, providing an efficient and tunable energy dissipation mechanism. The compact design, combined with the possibility of tailoring the mechanical response through spring stiffness adjustment, offers significant versatility for implementation in timber construction. Moreover, the simplicity of the analytical model ensures its practical applicability for design and engineering purposes.

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\* Corresponding author. Tel.: +393334858333.

E-mail address: [matteo.pellicciari@unimore.it](mailto:matteo.pellicciari@unimore.it)

## 1. Introduction

Slip-friction connectors dissipate energy through controlled sliding between preloaded surfaces, ensuring hysteretic behavior under cyclic loads. Due to their simplicity and effectiveness, they are increasingly adopted in structural engineering for seismic protection, especially in steel and concrete systems (Du et al., 2021; Jaisee et al., 2021; Dal Lago et al., 2017; Lu et al., 2023).

Recent research explores their use in timber engineering, where traditional mechanical fasteners often degrade under repeated loading, leading to reduced performance and difficult repair (Ceccotti et al., 2013; Zhang et al., 2021; Pei et al., 2016; Boggian et al., 2022). CLT panels and timber frame structures are commonly assembled using dry joints with metal fasteners, such as in hold-down systems, due to their low cost and ease of installation. Nevertheless, these conventional connections exhibit considerable deterioration when subjected to repeated cyclic loading, resulting in reduced stiffness, strength, and ductility over successive seismic events (Aloisio et al., 2022; Gavric et al., 2015). Furthermore, once damaged, these connections are challenging to repair or replace. In light of this, friction-based devices offer a promising alternative, showing improved energy dissipation and reusability compared to standard hold-downs (Loo et al., 2014; Hashemi and Quenneville, 2020; Hegeir et al., 2024).

Despite their potential, conventional slip-friction systems relying on bolt pretension face practical challenges: complex installation, preload loss over time, and surface wear, all of which may compromise performance (Golondrino et al., 2019). To overcome these issues, this study investigates an innovative slip-friction connector that uses a spring to generate normal force, eliminating the need for bolt pretension. The concept draws inspiration from the approach presented in (Dal Lago et al., 2021), applying the preload through a precompressed spring, with all components integrated within a hollow cylinder. The device is compact, versatile, and tunable in terms of stiffness and energy dissipation, making it suitable for seismic applications across various structural systems.

This work presents the device design, experimental testing, and an analytical model that captures its cyclic response. The proposed model is simple, closed-form, and validated against test data, providing a practical tool for engineering applications.

## 2. Experimental investigation

The proposed device, shown schematically in Fig. 1(a), consists of a hollow cylinder housing two hollow, wedge-shaped steel components, a precompressed spring, and a central bar. The lower wedge is fixed to the bar, while the upper wedge is free to move. The spring, compressed by tightening the lid, pushes the wedges together, activating friction at their inclined surfaces and at the lateral walls of the cylinder. When external displacement exceeds the frictional threshold, the wedges slide against the lateral walls, producing a stick-slip response with hysteretic energy dissipation. The spring provides an elastic restoring force, enabling the device to recover its initial configuration and sustain cyclic force-displacement behavior. This mechanism makes it suitable as a passive energy dissipation system in structures subjected to cyclic loading.

The wedges have a height of 60 mm, an outer diameter of 32 mm, and an inner diameter of 21 mm. The shear plane of the wedges is inclined at 60 degrees. The cylindrical housing has a total height of 110 mm, an external diameter of 37.5 mm, an internal diameter of 32.5 mm, and a wall thickness of 2.5 mm. The central threaded rod, to which the lower wedge is rigidly connected, has a diameter of 14 mm. At the top of the cylinder, a threaded section of 11.5 mm is used to secure the lid, ensuring the preload on the spring and maintaining the correct positioning of the internal components.

Three specimens with different springs were assembled for quasi-static cyclic testing (Pellicciari et al., 2025b). The threaded rod was connected to the lower wedge, while the spring was placed on the upper wedge. This assembly was inserted into the cylindrical housing and enclosed by tightening the lid. The specimens differed only in the spring type: (a) S1 with two coil springs in series (stiffness: 25 N/mm); (b) S2 with a single coil spring (stiffness: 75 N/mm); and (c) DS with a Belleville-type disc spring (initial stiffness: 246.7 N/mm). Further details on the springs are provided in (Pellicciari et al., 2025a).

Cyclic tests were performed using a ZwickRoell Z1200ES testing machine (Fig. 1(b)). The base of the connector was fixed to the machine using four M10 bolts, while the top was connected via a custom anchorage system for controlled displacement application. The loading protocol included three loading-unloading cycles from zero to a

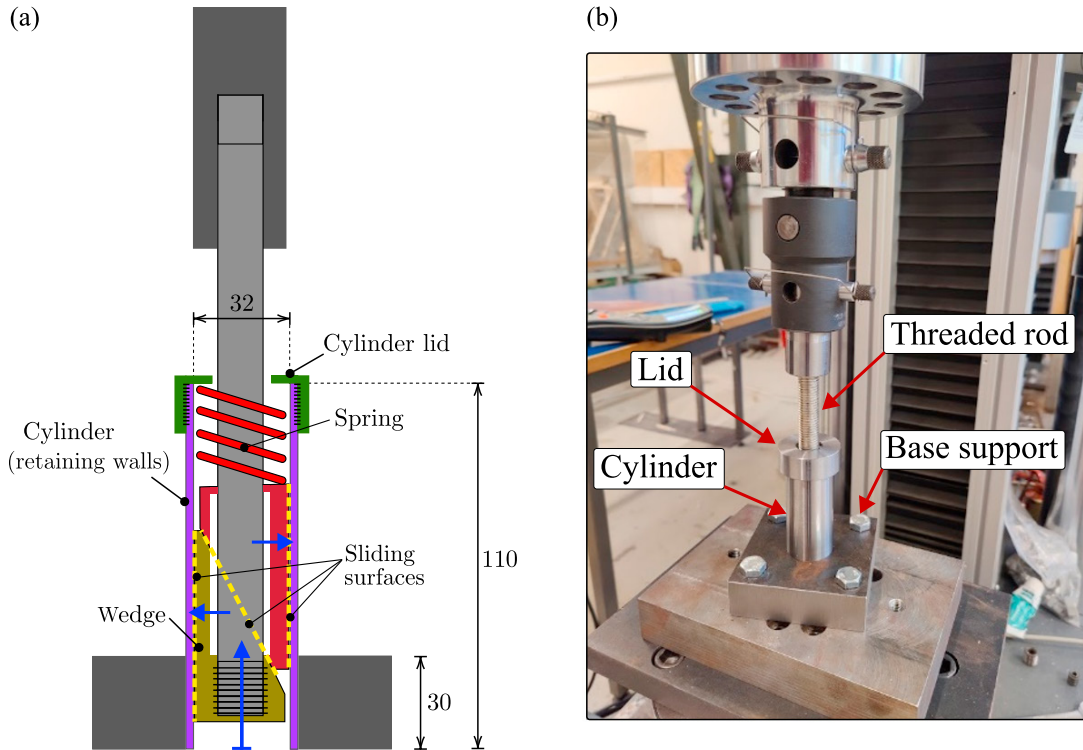


Fig. 1. (a) Schematic of the slip-friction device. The system comprises two wedges inside a hollow cylinder, preloaded by a compressed spring. The spring force presses the wedges against the inclined and lateral frictional interfaces. When the applied displacement exceeds the activation threshold, the wedges slide along the lateral walls, producing a stick-slip response with hysteretic force-displacement behavior due to combined elastic and frictional effects. Dimensions are in millimeters. (b) Experimental setup for quasi-static uniaxial cyclic tests used to evaluate the force-displacement response.

maximum displacement and back. A constant displacement rate of 5 mm/min ( $\approx 0.08$  mm/s) was applied in both directions to ensure quasi-static conditions. Prior to each test, the springs were preloaded to activate friction between the wedges and the cylinder walls in the initial configuration.

### 3. Analytical model

The proposed slip-friction device exhibits a characteristic stick-slip response under cyclic loading, governed by the interaction between frictional interfaces and the elastic restoring force of the preloaded spring. The device operates unidirectionally, with slipping phases occurring during both loading and unloading, as schematically illustrated in Fig. 2.

In the initial configuration, the spring is precompressed by a displacement  $\bar{x}_p$ , which generates a preload force  $F_p = F_s(\bar{x}_p)$  that activates friction at both the inclined wedge surfaces and the lateral walls of the housing. When an external displacement  $x$  is applied, the system remains in stick phase until the applied force reaches the activation threshold at  $x_A$ , beyond which slip initiates. The force continues to grow as the spring is further compressed, and upon reversal of displacement direction, the system transitions to unloading, exhibiting hysteresis due to the directional dependence of friction forces.

The mechanical response is defined by four main parameters: wedge inclination angle  $\theta$ , spring stiffness  $k$ , spring preload  $F_p$ , and friction coefficient  $\mu$ . Assuming Coulomb friction (Márton and Lantos, 2009), the force-displacement

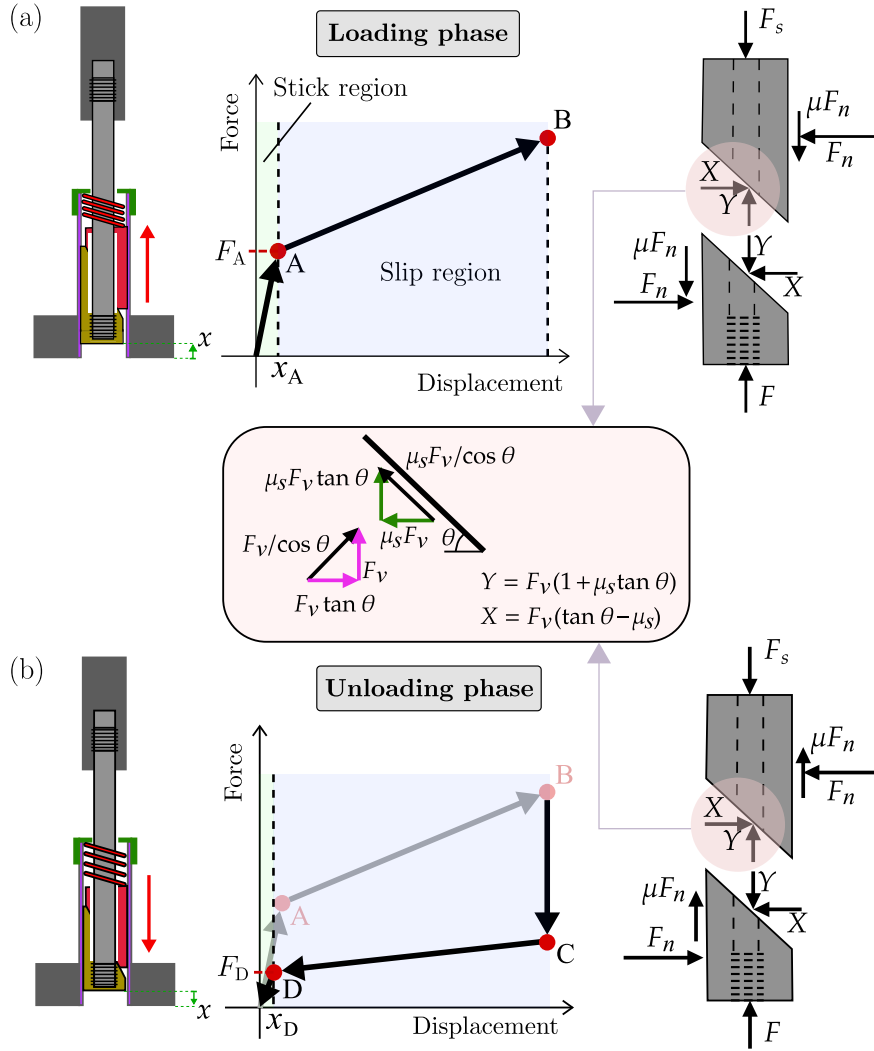


Fig. 2. Schematic of the slip–friction device under cyclic loading. Preload  $\bar{x}_p$  compresses the spring, pressing the wedges against the lateral walls and activating friction. (a) During loading, the device remains in stick region until  $x = x_A$ , then slips as the activation force is exceeded, with force increasing to point B. (b) Reversing displacement starts unloading, with a force drop to point C. The system returns to stick region at  $x = x_D$ .

relationship of the device is expressed as:

$$F = \begin{cases} k_A x, & \text{for } \dot{x} > 0 \text{ and } x \leq x_A, \\ \psi F_s (\bar{x}_p + x - x_A), & \text{for } \dot{x} > 0 \text{ and } x > x_A, \\ \psi^{-1} F_s (\bar{x}_p + x - x_D), & \text{for } \dot{x} < 0 \text{ and } x > x_D, \\ k_D x, & \text{for } \dot{x} < 0 \text{ and } x \leq x_D, \end{cases} \quad (1)$$

where the directional scaling factor  $\psi$  is defined as

$$\psi = \frac{1 - \mu_s \mu + (\mu_s + \mu) \tan \theta}{1 + \mu_s \mu + (\mu_s - \mu) \tan \theta}, \quad (2)$$

with  $\mu_s$  being the static friction coefficient and  $\mu$  the velocity-dependent friction coefficient, defined, for instance, by the Stribeck model  $\mu = \mu_k + (\mu_s - \mu_k)e^{-\dot{x}/v_s}$ , where  $\dot{x} = dx/dt$ ,  $\mu_k$  is the kinetic friction coefficient, and  $v_s$  is a reference velocity (Xiao et al., 2024). The stiffnesses  $k_A$  and  $k_D$  in the stick regions during loading and unloading are given by

$$k_A = k_n \left( \mu_s + \mu + \frac{1 + \mu_s^2}{\tan \theta - \mu_s} \right), \quad (3)$$

$$k_D = k_n \left( \mu_s - \mu + \frac{1 + \mu_s^2}{\tan \theta - \mu_s} \right), \quad (4)$$

where  $k_n$  accounts for the deformation of the lateral walls of the cylinder:

$$k_n = \frac{4\pi\bar{E}J_z(\tau hR^2 + J_z)\cot\theta}{(\pi^2 - 8)\tau hR^5 + \pi^2 J_z R^3}. \quad (5)$$

Here,  $R$  is the mean radius of the hollow cylinder,  $\tau$  its wall thickness,  $h$  the height of the contact pressure area, and  $\bar{E} = E/(1 - \nu^2)$  the effective Young's modulus, with  $\nu$  being Poisson's ratio. The term  $J_z$  is given by  $J_z = -\tau hR^2 - hR^3 \ln((2R - \tau)/(2R + \tau))$ .

This model captures the asymmetric hysteretic force–displacement behavior of the device, combining frictional and elastic contributions. The reader is referred to (Pellicciari et al., 2025a) for full details of the model derivation.

### 3.1. Limit case of rigid retaining cylinder walls

In practice, the sliding displacements  $x_B$  (or  $x_C$ ) are much larger than the stick-region amplitudes  $x_A$  and  $x_D$ , which stem from the deformation of the retaining walls. Assuming the cylinder behaves as rigid, these displacements can be neglected ( $x_A, x_D \rightarrow 0$ ), and Eq. (1) simplifies to:

$$F = \begin{cases} \psi F_s, & \dot{x} > 0, \\ \psi^{-1} F_s, & \dot{x} < 0, \end{cases} \quad (6)$$

where  $F_s = F_s(\bar{x}_p + x)$  and  $\psi$  is defined in Eq. (2). This simplified model is practical, reducing both implementation effort and mathematical complexity, and can be readily applied in timber structures.

## 4. Results

The analytical model is validated against quasi-static cyclic tests on specimens S1, S2, and DS. The static friction coefficient is set to  $\mu_s = 0.45$ , consistent with typical steel-on-steel contact (Xie et al., 2000; Pijpers and Slot, 2020). The low test displacement rate (quasi-static) justifies the assumption  $\mu \approx \mu_s$  in the model. Preload displacements  $\bar{x}_p$  and corresponding forces  $F_p$  are summarized in Table 1, along with other model parameters.

Figure 3 compares experimental force–displacement and energy curves (dashed lines) with model predictions from Eq. (1) (solid lines). The model captures well the cyclic behavior, including stiffness in both phases and activation forces. Even under the simplifying assumption of rigid retaining walls (Eq. (6)), predictions would remain accurate, as  $x_A$  and  $x_D$  are negligible compared to the sliding displacement.

The largest discrepancy appears in DS during unloading (Fig. 3(e)), likely due to higher variability and fewer repeated tests. Nonetheless, the model shows robust performance and good agreement overall, despite assumed rather

Table 1. Parameters of the analytical model and experimental setup for specimens S1, S2, and DS.

| Parameter                   | Symbol      | S1   | S2   | DS    |
|-----------------------------|-------------|------|------|-------|
| Wedge angle                 | $\theta$    | 60°  | 60°  | 60°   |
| Static friction coefficient | $\mu_s$     | 0.45 | 0.45 | 0.45  |
| Spring stiffness (N/mm)     | $k$         | 25   | 75   | 246.7 |
| Preload displacement (mm)   | $\bar{x}_p$ | 9    | 5    | 3     |
| Preload force (N)           | $F_p$       | 225  | 375  | 675.5 |

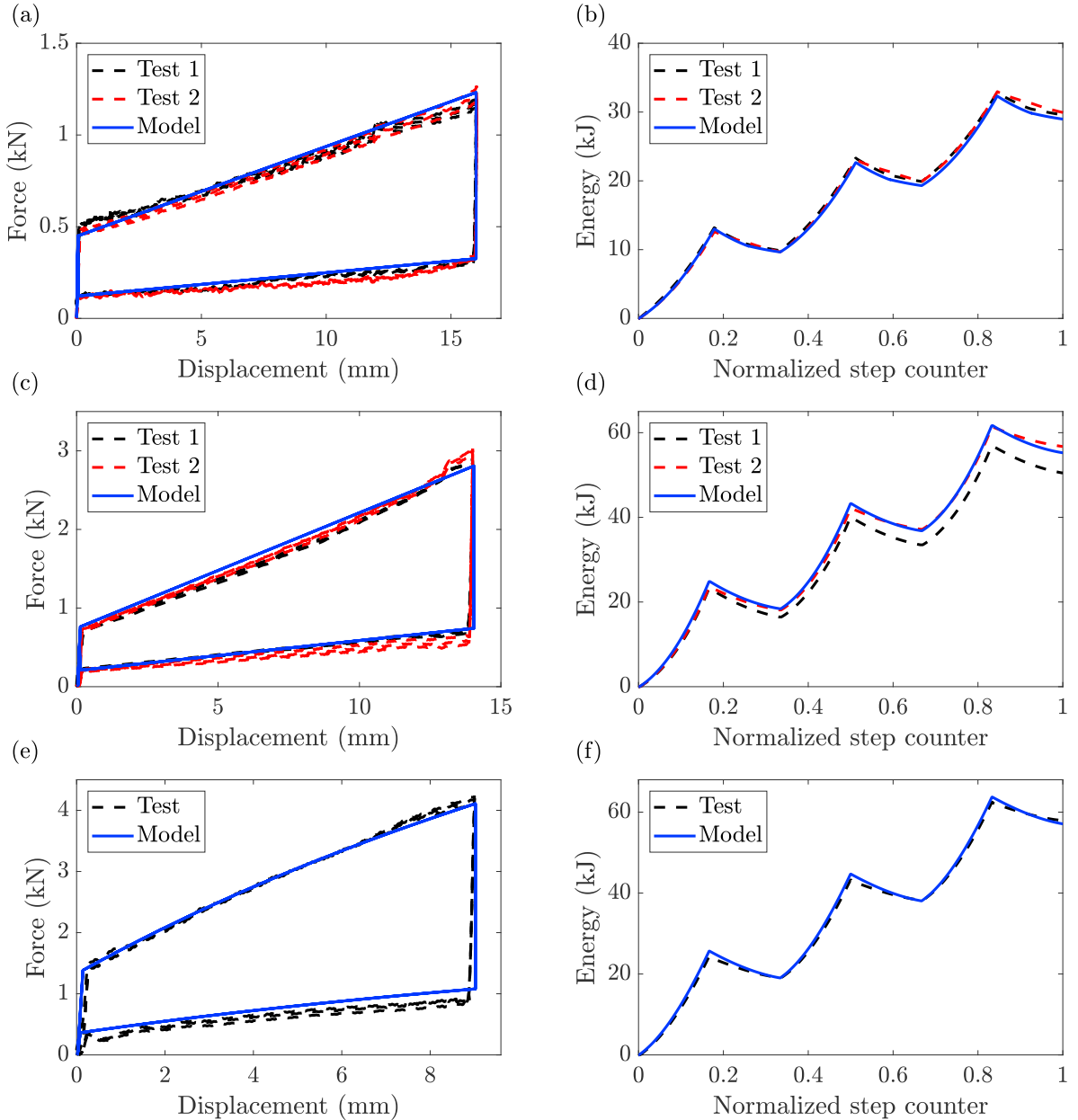


Fig. 3. Experimental (dashed) and analytical (solid) force–displacement and total input energy curves for specimens: (a,b) S1, (c,d) S2, and (e,f) DS.

than measured friction values. In summary, the proposed analytical model provides an effective and accurate representation of the cyclic response, demonstrating its validity for predicting the mechanical behavior of the slip–friction connector.

## 5. Conclusions

This study examined the mechanical behavior of a novel slip–friction connector for structural applications. The device addresses common limitations of conventional friction-based connections, such as preload loss, complexity, and limited long-term stability, by combining frictional sliding with an elastic restoring force from a spring, eliminating the need for bolt pretension. Analytical modeling and experimental testing were integrated to characterize its cyclic response and assess practical feasibility.

The analytical model, derived from equilibrium equations, provides a closed-form solution that accurately predicts the hysteretic behavior of the device. It captures the interplay between frictional slip and elastic restoring forces, and highlights the influence of design parameters on performance and energy dissipation. Its simplicity and accuracy make it suitable for engineering applications.

Experimental tests validated the model, showing strong agreement with predictions and confirming the stability, repeatability, and energy dissipation capacity of the device. The ability to tune its response by adjusting the spring, without altering the overall geometry, further enhances its versatility.

The results demonstrate the potential of the proposed connector as an effective alternative to traditional dissipative connections, particularly in timber structures, but also in other systems requiring controlled energy dissipation. Its advantages include ease of implementation, compact design, self-centering capability, and adaptability. Beyond structural engineering, the concept shows promise for applications in vibration mitigation, mechanical damping (e.g., aerospace and impact absorption), vehicle suspensions, and similar systems.

This study focused on quasi-static conditions. Future work will include a comprehensive dynamic experimental campaign and model refinements to capture advanced friction behavior, wear, and contact degradation. Further simulations under realistic cyclic excitations, such as seismic, harmonic, and impact loads, will also be conducted to explore broader applications.

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